

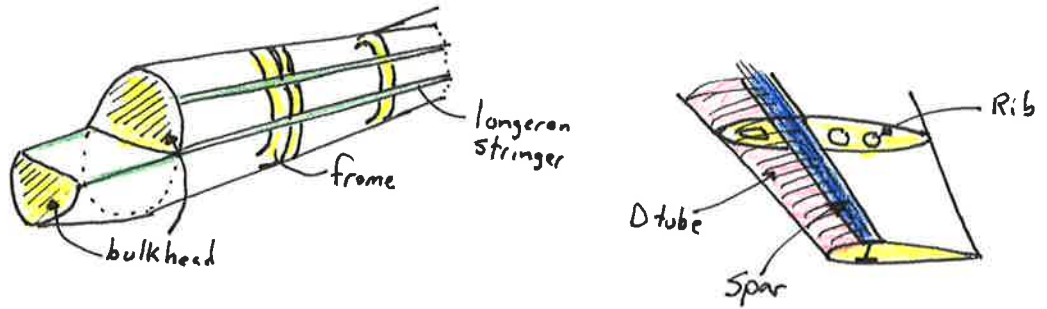
Review for Exam 1

W 14th
St. Valentines Day

Do you really
♥ aero structures?!

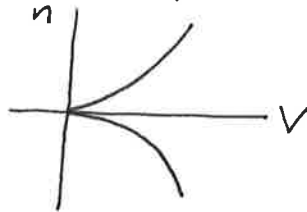
1 Motivation

- Strong, lightweight, robust volumes and areas with inertial and aerodynamic loads.
- Terminology



2 Loads

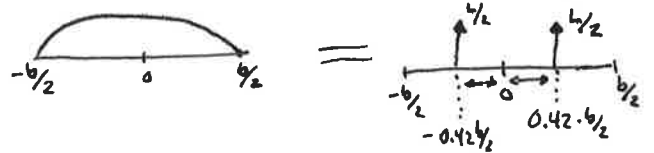
- Loads are the inertial, aero, etc forces and pressures applied to the structure during operation
- Aero loads scale with dynamic pressure $\equiv \frac{1}{2} \rho V^2$ at low speeds



This is why the stall line of a V-n diagram is a parabola: $n \propto V^2$

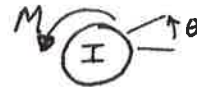
- The lift distribution is a measure of how the lift forces/pressure are spanwise loaded. Elliptical is a classic distribution, but is not always true... depends on geometry

$$L' = \frac{L}{2} \frac{2}{b} \frac{4}{\pi} \sqrt{1 - \left(\frac{2x}{b}\right)^2}$$



3 Inertial Loading

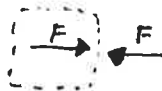
$$\vec{F} = m\vec{a}$$



$$I\ddot{\theta} = M$$

Equal and opposite reaction (on either side of control volume)

$$I = \int r^2 dm$$



Free body diagram

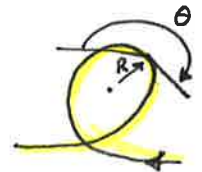
Dynamics:

$$M = r \times F$$

$$a_{pt} = a_{xyz} + \ddot{\theta} \times \vec{r} + \dot{\theta} \times (\dot{\theta} \times \vec{r}) + 2\dot{\theta} \times \vec{v}$$

A level banked turn gives a g-load of $n = \frac{L}{W} = \frac{1}{\cos \phi}$

A steady state pitching maneuver $n_z = \cos \theta + \frac{V^2}{R} \frac{1}{g}$



Static column of fluid $\frac{dp}{dz} = -\rho gh$

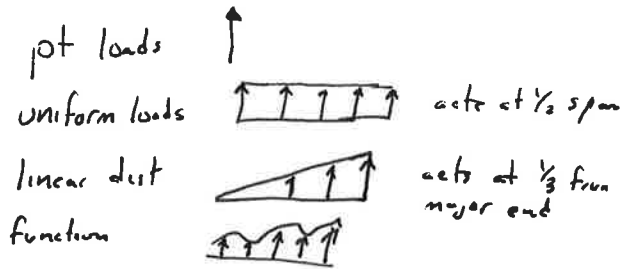
4 Gust Loading
Concepts, not much calculation in this class.

5 Shear and Moment Diagrams

- Find wall reaction shear and moment
 - Cut at desired point
- If in doubt, integrate

$$V = \int (F \delta(x-x_i) + F') dy$$

$$M = \int (F \delta(x-x_i) + F') y dy$$



6 Beam properties

$$\text{Area} = \int dA$$

$$\bar{y} \text{ Centroid} = \frac{1}{A} \int y dA$$

$$I_{yy} = \int (z')^2 dA$$

7 Beam Bending



8 Traction Vector

$$T(v) = \frac{dF}{dA} \text{ in a direction } v$$

$$\sigma_n = T \cdot v$$

$$\sigma_s = |T - (T \cdot v)v|$$

Cauchy Formula relates surface tractions to stresses

$$T_x = \sigma_{xx} v_x + \sigma_{yx} v_y + \sigma_{zx} v_z$$

Can be used to find normal + shear in direction

9 Stress Transformation

$$\sigma' = A \sigma A^T \text{ with } A_{ij} = e'_i \cdot e_j$$

10 Principal Stresses

Eigenvalues of $[\sigma]$ give $\sigma_1, \sigma_2, \sigma_3$

Eigenvectors of $[\sigma]$ give directions

shear is $\frac{1}{2}$ difference of σ_1, σ_3

11 von Mises
Yield criterion

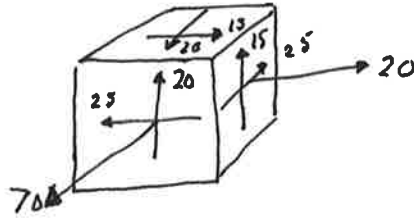
12 Anisotropic materials $[\sigma] = [D][\epsilon] - [I]\alpha \Delta T$

2D loading is a special case of 3D

Non-isotropic materials

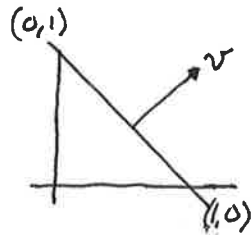
Quiz

Determine if the following stress state is safe. The material fails at $\sigma_n = 80$ ksi



Quiz

Given the stress state $\sigma = \begin{bmatrix} 10 & 8 \\ 8 & 20 \end{bmatrix}$ determine the traction vector on the plane A-B



Some Useful Equations + Properties

$$F = ma$$

$$I \ddot{\theta} = M$$

$$I = \int r^2 \rho dx dy dz = m R_g^2$$

$$I_o = I_{cg} + M \bar{x}^2$$

$$a_{pt} = a_{xyz} + \ddot{\theta} \times r + \dot{\theta} \times (\dot{\theta} \times r) + 2 \dot{\theta} \times v_{relative}$$

$$g_o = 32.174 \frac{ft}{s^2}$$

Constant acceleration: $v = at$ $s = \frac{1}{2} at^2 = \frac{1}{2} \frac{v^2}{a}$

$$Power = F \cdot v$$

$$\text{Jet } A \rho \approx 6.7 \frac{lb}{gal}$$

$$|g_{ad}| = 0.133 \frac{ft^3}{s^3}$$

$$7.4 \text{ gal} = 1 \text{ ft}^3$$

$$\frac{dP}{dz} = -\rho g$$

$$\rho_{H_2O} = 8.34 \frac{lb}{gal}$$

$$\bar{y} = \frac{1}{A} \int y dA \quad \bar{z} = \frac{1}{A} \int z dA$$

$$I_{yy} = \int (z')^2 dA \quad I_{yz} = \int y'z' dA$$

$$I_{y'y'} = I_{yy} + z^2 A$$

$$\sigma_{xx} = -\frac{My}{I}$$

$$\frac{d^2 v}{dx^2} = \frac{M}{EI}$$

$$T(v) = \frac{dF}{dA}$$

$$\sigma_N = T \cdot v$$

$$\sigma_s = |T - (T \cdot v)v|$$

$$\sigma' = A \sigma A^T \text{ with } A_{ij} = e'_i \cdot e_j$$

$$T_x = \sigma_{xx} v_x + \sigma_{yx} v_y + \sigma_{zx} v_z$$

$$T_y = \sigma_{xy} v_x + \sigma_{yy} v_y + \sigma_{zy} v_z$$

$$T_z = \sigma_{xz} v_x + \sigma_{yz} v_y + \sigma_{zz} v_z$$

$$-\sigma_p^3 + I_1 \sigma_p^2 - I_2 \sigma_p + I_3 = 0$$

Equilibrium

$$I_1 = \sigma_{xx} + \sigma_{yy} + \sigma_{zz}$$

$$I_2 = \begin{vmatrix} \sigma_{xx} & \sigma_{xy} \\ \sigma_{xy} & \sigma_{yy} \end{vmatrix} + \begin{vmatrix} \sigma_{yy} & \sigma_{yz} \\ \sigma_{yz} & \sigma_{zz} \end{vmatrix} + \begin{vmatrix} \sigma_{zz} & \sigma_{xz} \\ \sigma_{xz} & \sigma_{xx} \end{vmatrix}$$

$$I_3 = \det[\sigma]$$

$$\frac{d\sigma_{xx}}{dx} + \frac{d\sigma_{xy}}{dy} + \frac{d\sigma_{xz}}{dz} + X = 0$$

$$\sigma_{ij} = \sigma_{ji}$$

Von-Mises

$$\frac{1}{6} \left[(\sigma_{xx} - \sigma_{yy})^2 + (\sigma_{yy} - \sigma_{zz})^2 + (\sigma_{xx} - \sigma_{zz})^2 + 6\sigma_{xy}^2 + 6\sigma_{xz}^2 + 6\sigma_{xy}^2 \right] \leq \frac{Y^2}{3}$$

Isotropic

iso 3D

$$(\sigma) = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & \nu & & & \\ \nu & 1-\nu & \nu & & & \\ \nu & \nu & 1-\nu & & & \\ & & & 1-2\nu & & \\ & & & & \dots & \end{bmatrix} (\epsilon)$$

$$\begin{pmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{pmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & & \\ \nu & 1 & & \\ & & 1-\nu & \\ & & & \dots \end{bmatrix} \begin{pmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{pmatrix}$$

$$(\epsilon) = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu & & & \\ -\nu & 1 & -\nu & & & \\ -\nu & -\nu & 1 & & & \\ & & & 2(1+\nu) & & \\ & & & & \dots & \end{bmatrix} (\sigma)$$

$$(\epsilon) = \frac{1+\nu}{E} \begin{bmatrix} 1-\nu & -\nu & & \\ -\nu & 1-\nu & & \\ & & 2 & \\ & & & 1-2\nu \end{bmatrix} (\sigma)$$

$$(\epsilon) = +\alpha \Delta T \quad \Leftrightarrow \quad (\sigma) = -\frac{\alpha \Delta T E}{1-2\nu}$$

Ex: A fairing has an aerodynamic load of 0.5 psi at 200 mph. What is the velocity of failure if the fairing has an ultimate load of 3 psi?

$$\underbrace{\Delta P}_{\text{aero load}} = p - p_{\infty} = \frac{1}{2} \rho V^2 C_p \Rightarrow \frac{\Delta P_1}{\Delta P_2} = \frac{\frac{1}{2} \rho V_1^2 C_p}{\frac{1}{2} \rho V_2^2 C_p} = \frac{V_1^2}{V_2^2}$$

• 0.5 psi at 200 mph

$$V_2^2 = V_1^2 \frac{\Delta P_2}{\Delta P_1} = \cancel{200 \text{ mph}} \Rightarrow V_2 = V_1 \sqrt{\frac{\Delta P_2}{\Delta P_1}}$$

$$V_2 = 200 \text{ mph} \left(\frac{3.0 \text{ psi}}{0.5 \text{ psi}} \right)^{\frac{1}{2}} = \boxed{490 \text{ mph} = V_2}$$

6x higher ~~speed~~ pressure with only 2.4x higher speed

Ex: Determine the root moment of an elliptical wing of span 34 ft generating 6000 lbf of lift.

Hard way: Integrate the lift distribution

$$M = \int dM dy = \int L' y dy = \int_0^{b/2} \frac{L}{2} \frac{2}{b} \frac{4}{\pi} \sqrt{1 - \left(\frac{2y}{b}\right)^2} dy = \dots$$

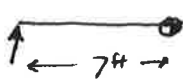
Easy: Remember Elliptical acts at 42% semispan (exactly 42.44%)

$$M = F \cdot \text{distance} = \left(\frac{L}{2}\right) \left(\frac{b}{2}\right) (0.42)$$

$$= \frac{3000 \text{ lbf} \cdot 17 \text{ ft}}{0.4244} = \boxed{21644 \text{ ft-lbf}}$$

Ex: A student pilot lands poorly on the nosegear. of a Piper Cherokee. The nose wheel is 7 ft forward of the CG. The aircraft has a weight of 1500 lb and a radius of gyration of 9 ft. A g meter reads an instantaneous +3g. Determine the pitch acceleration and the time to reach $15^\circ = \theta$ if the aircraft "lands" at $\theta = -5^\circ$

① FBD



$$I = m R_g^2 = \frac{1500 \text{ lbf}}{32.174 \text{ ft/s}^2} \left| \frac{9 \text{ ft}}{1 \text{ ft}} \right|^2 = 3776 \text{ slug-ft}^2$$

② g-load

$$F = ma = \frac{1500 \text{ lbf}}{32.174 \text{ ft/s}^2} \left| \frac{3g}{1g} \right| = 4500 \text{ lbf}$$

$$M = r \times F = 7 \text{ ft} \cdot 4500 \text{ lbf} = 31500 \text{ ft-lbf}$$

③ Acceleration

$$\ddot{\theta} = \frac{M}{I} = \frac{31500 \text{ ft-lbf}}{3776 \text{ slug-ft}^2} \left| \frac{\text{slug-ft}}{\text{lbf-s}^2} \right| = \boxed{8.3 \frac{\text{rad}}{\text{s}^2}}$$

Handwritten scribbles and notes, including a circled '8.3' and some illegible text.

④ Time $\theta = \frac{1}{2} \ddot{\theta} t^2 \Rightarrow t = \sqrt{\frac{2\theta}{\ddot{\theta}}} = \sqrt{\frac{2 | 20^\circ | \frac{s^2}{rad} \pi}{8.3 \frac{rad}{s^2} | 180^\circ}} = \underline{0.3 s}$
 Could the pilot react?

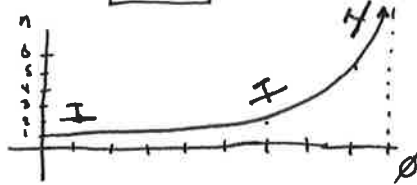
⑤ Reality

The front gear will collapse.

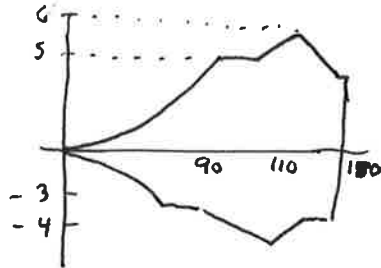
tiny.cc/NMMHYWjEmKY

Ex: What is the g-load of an aircraft in a steady 45° bank?

$n = \frac{1}{\cos 45^\circ} = \boxed{1.41}$ Acceleration = $ng = 1.41 \cdot 32.174 = 45.5 \frac{ft}{s^2}$



Ex: Determine the limit load g load and VNE from the V-n diagram -



$\boxed{\begin{matrix} \text{g-load } +6 \\ V_{NE} \quad 180 \end{matrix}}$

Ex: A 140 lb pilot performs a 1000 ft radius loop at 200 ft/s. What is the g-load when going vertical up?



$\theta = 90^\circ$

$n_2 = \cos 90^\circ + \frac{200^2 \frac{ft^2}{s^2}}{8^2 | 1000 \frac{ft}{s^2} | 32.174 \frac{ft}{s^2}}$

$\boxed{n = 1.24}$

Ex: ~~Answer~~ A rocket accelerates at 5g. If a 100 ft tank is full of Jet A, what is the pressure at the tank's bottom?

$\frac{dp}{dz} = -\rho g \Rightarrow dp = -\rho g dz \Rightarrow \Delta p = -\rho g \Delta z$

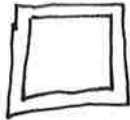
$\Delta p = - \frac{6.7 \frac{lb}{ft^3} | 32.174 \frac{ft}{s^2} | 100 \frac{ft}}{ft^3} | \frac{7.4 \frac{gal}{ft^3} | 5g}{ft^3} | \frac{1 \frac{ft^3}{gal}}{32.174 \frac{ft}{s^2} | 144 \frac{in^2}{ft^2}}$

Need mass not weight!

$\boxed{= 172 \text{ psi}}$

Ex: Given a square tube spec, determine the deflection due to a uniform loading of 1000 lbs over 24 ft with a 2 inch OD and $\frac{1}{8}$ in wall thickness

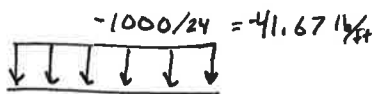
① Beam



$$I_{yy_{outer}} = \frac{1}{12} b h^3 \quad I_{yy_{inner}} = \frac{1}{12} b_i h_i^3$$

$$I_{yy} = \frac{1}{12} (2)(2)^3 - \frac{1}{12} (1.75)(1.75)^3 = 0.55 \text{ in}^4$$

② loading



③ Wall loading



$$V_0 + \int_0^x (-41.67) dx = 0 \quad V_0 = 1000 \text{ lbf}$$

$$M_0 + \int_0^x (+41.67) x dx = 0 \quad M_0 = -12000 \text{ ft-lbf}$$

④ Shear + Moment



$$V_0 - \int_0^x 41.67 dx + V = 0 \quad V(x) = -V_0 - 41.67 x = -1000 - 41.67 x$$

$$M_0 + \int_0^x (41.67) x dx - M(x) = 0 \quad M(x) = M_0 - 41.67 \frac{x^2}{2} = -12000 \text{ ft-lbf} - 20.83 x^2$$

⑤ deflection

$$\frac{d^2V}{dx^2} = \frac{M}{EI} = \frac{-12000 - 20.83 x^2}{EI}$$

$$V = \frac{1}{EI} \left(-12000 \frac{x^2}{2} - \frac{20.83}{3.4} x^4 \right)$$

$$= \frac{1}{10.5 \times 10^6 \text{ psi} \cdot 0.55 \text{ in}^4} \left(-12000 \frac{24^2}{2} - \frac{20.83}{3.4} 24^4 \right) \frac{\text{ft-lbf-ft}^2}{\text{psi} \cdot \text{in}^4} \cdot \frac{144 \text{ in}^2}{\text{ft}^2}$$



100 ft

Why does this not make sense?

⑥ Stress?

$$\sigma_{xx} = -\frac{M_y}{I} = \frac{-12000 \text{ ft-lb} \cdot 1 \text{ in}}{0.55 \text{ in}^4 \cdot \text{ft}} = 261 \text{ ksi}$$

Ex: A 3D orthotropic material is loaded as

$$[\sigma] = \begin{bmatrix} 10 \text{ ksi} & 2 \text{ ksi} & 0 \\ 2 \text{ ksi} & 5 \text{ ksi} & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

Determine ϵ_{xx} if $\Delta T = 200^\circ\text{F}$ and $E_x = 25 \times 10^6 \text{ psi}$

$$E_y = 2 \times 10^6 \text{ psi}$$

$$\nu = 0.3$$

$$\alpha = 10 \times 10^{-6} \text{ } / ^\circ\text{F}$$

$$\textcircled{1} \quad \epsilon_{xx} = \frac{1}{E_x} \sigma_{xx} - \frac{\nu_{xy} \sigma_{yy}}{E_x} - \frac{\nu_{xz} \sigma_{zz}}{E_x} + \alpha \Delta T$$

$$= \frac{1}{25 \times 10^6 \text{ psi}} (10 \times 10^3 - 0.3 \cdot 5 \times 10^3 - 0) + 10 \times 10^{-6} \cdot 200$$

$$\boxed{\epsilon_{xx} = 0.00234}$$

Ex: Sunken Ship

$$p = \rho g h = \frac{8.34 \text{ lbf}}{\text{gal}} \cdot \frac{\text{slugs}}{32.174 \text{ lbf}} \cdot \frac{\text{ft}}{0.133 \text{ ft}^3} \cdot \frac{32.174 \text{ ft}}{\text{ft}} \cdot \frac{12500 \text{ ft}}{5.105 \text{ ft}} \cdot \frac{\text{ft}^2}{\text{slugs} \cdot \text{ft}} \cdot \frac{\text{ft}^2}{144 \text{ in}^2}$$

$$= 5400 \text{ psi}$$

$$\sigma = \begin{bmatrix} 5.4 & & \\ & 5.4 & \\ & & 5.4 \end{bmatrix} \times 10^3 \text{ psi}$$

Von Mises

$$\frac{1}{6} \left((5.4 - 5.4)^2 + (\dots)^2 + (\dots)^2 + (\dots)^2 \right) \leq \frac{Y^2}{3} \quad \checkmark \text{ yes, always}$$

$\boxed{\text{Not yielded}}$

Solids deflect but do not yield or fail due to the ocean's pressure.

$\boxed{\text{Ships are intact}}$... getting the ship out of the ocean floor might be difficult