

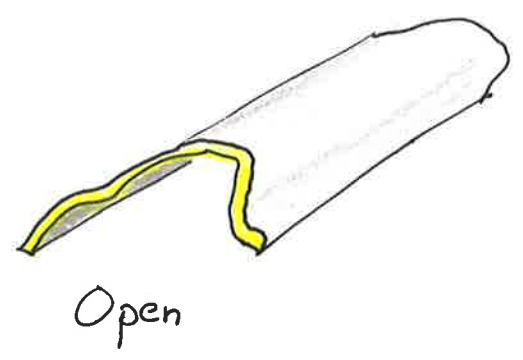
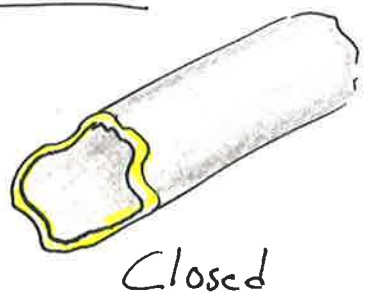
Torsion in Thin Walled Sections

What about m_x ?



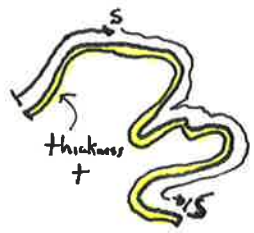
$$\frac{dM_x}{dx} = -m_x(x)$$

Types of sections



Membrane Analysis for ^{thin homogeneous} Open Sections ONLY

(do not use for closed sections!)
 do not use for multi material sections.
 do not use for thick sections (i.e. σ_{xs} uniform) across section wall

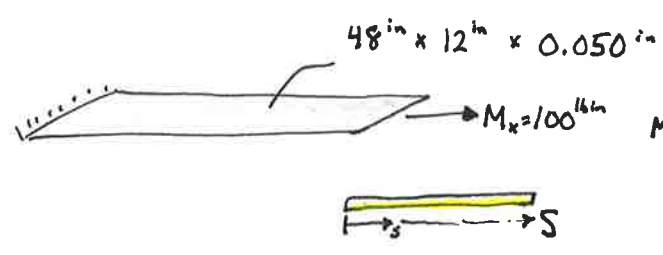


$$J = \frac{st^3}{3}$$

Maximum shear stress

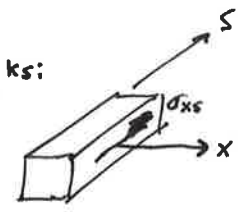
$$\sigma_{xs} = \frac{M_x t}{J} = \frac{M_x 3}{st^2}$$

Ex:

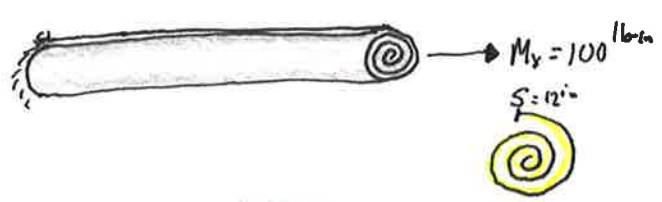


Maximum shear stress

$$\sigma_{xs} = \frac{100 \text{ lb-in} \cdot 3}{12 \text{ in} \cdot 0.050 \text{ in}^2} = 10 \text{ ksi}$$



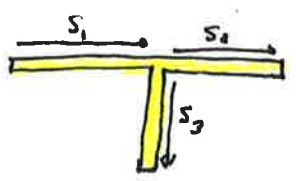
Ex: Roll up the above sheet



$$\sigma_{xs} = (\dots) = 10 \text{ ksi}$$

The configuration does not matter!

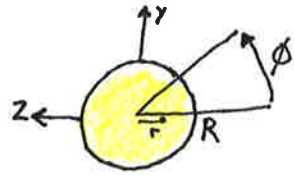
Multisection Open sections



$$J = \sum \frac{S_i t^3}{3} = \frac{S_1 t^3}{3} + \frac{S_2 t^3}{3} + \frac{S_3 t^3}{3}$$

$$\sigma_{xs} = \frac{M_x t}{J}$$

Circular Bar Torsion



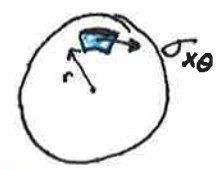
$$\epsilon_{x\theta} = r \frac{\phi(L)}{L}$$

$$\epsilon_{x\theta} = \frac{R \phi(L)}{L}$$

$$\Rightarrow \epsilon_{x\theta}(r) = \frac{r}{R} \epsilon_{x\theta}(b) \Rightarrow \sigma_{x\theta} = \underbrace{G}_{\text{shear modulus}} \epsilon_{x\theta}$$

$$\sigma_{x\theta}(r) = \frac{r}{R} \sigma_{x\theta}(R)$$

Static equilibrium



$$dA = r dr d\theta$$

$$\sum M_x = 0 = -M_x + \int_A \sigma_{x\theta} r dA \Rightarrow M_x = \int_0^{2\pi} \int_{R_{inner}}^{R_{out}} \underbrace{\frac{r}{R_{out}} \sigma_{x\theta}(R)}_{\text{stress}} \underbrace{r}_{\text{distance}} \underbrace{r dr d\theta}_{dA}$$

define $J = \text{polar moment of inertia} = \int_0^{2\pi} \int_{R_{in}}^R r^3 dr d\theta$

$$M_x = \frac{\sigma_{x\theta}(R)}{R} \int_0^{2\pi} \int_{R_{in}}^{R_{out}} r^3 dr d\theta = \frac{\sigma_{x\theta}}{R} J$$

solve for $\sigma_{x\theta}$

$$\sigma_{x\theta}(R) = \frac{M_x R}{J} \quad \text{substitute above} \quad \sigma_{x\theta}(r) = \frac{r}{R} \sigma_{x\theta}(R) = \frac{M_x R}{J} \frac{r}{R}$$

$$\boxed{\sigma_{x\theta}(r) = \frac{M_x r}{J}}$$

with a rotation angle $\boxed{\phi(x) = \frac{M_x x}{JG}}$

Ex: Thin tube in torsion



$$S = 2\pi R t$$

$$J = \int_0^{2\pi} \int_{R-t}^R r^3 dr d\theta = \int_0^{2\pi} \left[\frac{r^4}{4} \right]_{R-t}^R d\theta = \frac{r^4}{4} 2\pi \Big|_{R-t}^R \approx \frac{\pi}{2} (4R^3 t - 6R^2 t^2 + 4R t^3)$$

higher order terms H.O.T

= $2\pi R^3 t$ compare with table in book.

$$\sigma_{x\theta_{max}} = \frac{M_x R}{J} = \frac{M_x}{2\pi R^2 t} = \frac{M_x}{S R t} = \frac{M_x 2\pi}{S^2 t} \quad \text{Compare with open section of same } S !!$$

4.8 Thermal

① Temperature $\Delta T = T_2 - T_1 = (-110^\circ\text{F}) - (90^\circ\text{F}) = -200^\circ\text{F}$ Brrr.....

② Heterogeneous beam with Thermal loads (4.55)

$$\sigma_{xx} = \frac{E}{E_1} \frac{P + P^T}{A^*} + \underbrace{\dots}_{0} y + \underbrace{\dots}_{0} z - E \alpha \Delta T$$

$$P^T = \int E \alpha \Delta T dA = \sum E_i \alpha_i \Delta T_i A_i \quad \text{and} \quad M_y^T = \sum E \alpha \Delta T \bar{z} A$$

③ Calculate P^T

$$\begin{aligned} P^T &= 30 \times 10^6 \cdot 6.78 \times 10^{-6} \cdot (-200) \cdot 12 \cdot 0.4375 \quad \text{steel} \\ &+ 10 \times 10^6 \cdot 13 \times 10^{-6} \cdot (-200) \cdot 10 \cdot 7 \cdot 0.1 \quad \text{aluminum} \\ &= -395570 \text{ lbf} \end{aligned}$$

but this is a symmetric beam about the centroid!

④ Find A^*

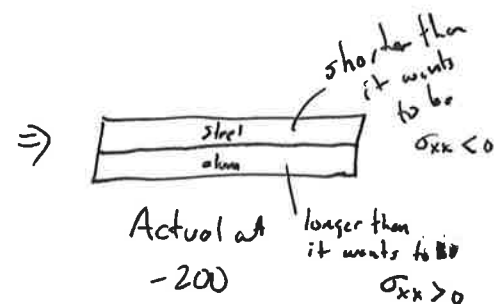
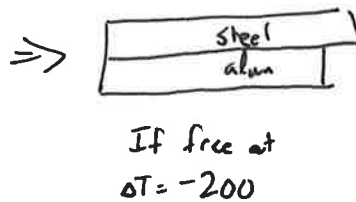
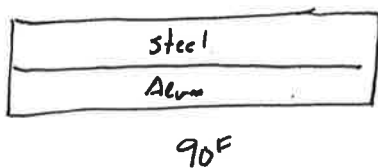
From earlier, $A^* = 22.75 \text{ in}^2$

⑤ Stress

$$\begin{aligned} \sigma_{xx \text{ steel}} &= \frac{30}{10} \cdot \frac{-395570}{22.75} - 30 \times 10^6 \cdot 6.78 \times 10^{-6} \cdot (-200) \\ &= -11.5 \text{ ksi} \end{aligned}$$

$$\begin{aligned} \sigma_{xx \text{ al}} &= \frac{1}{1} \cdot \frac{-395570}{22.75} - 10 \times 10^6 \cdot 13 \times 10^{-6} \cdot (-200) \\ &= 8.6 \text{ ksi} \end{aligned}$$

⑥ Visual



Nuts

Goes on the threaded portion of the bolt.....

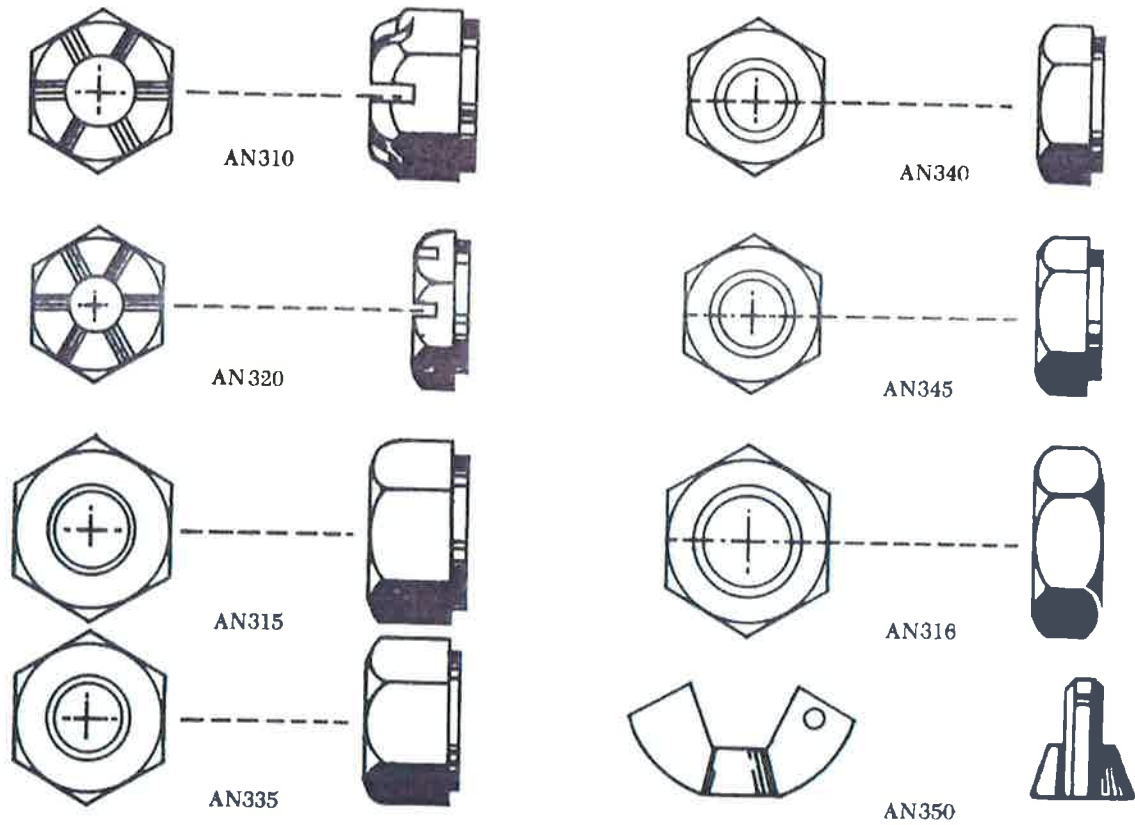
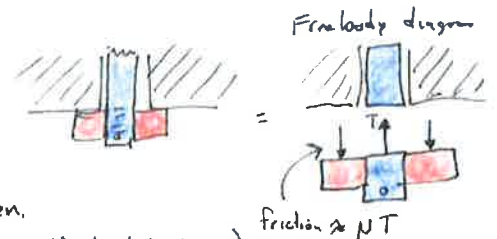


FIGURE 6-6. Non-self-locking nuts.

Non-Self locking

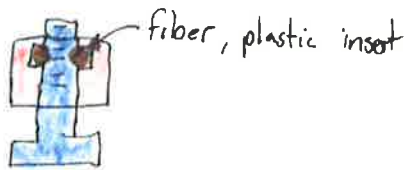
Loosening rotation prevented only by tightness + friction
like on airplane

In a high vibration environment, these nuts will loosen.
Must be safety wired or pinned. (Hence AN3-5 bolt with drilled shank!)

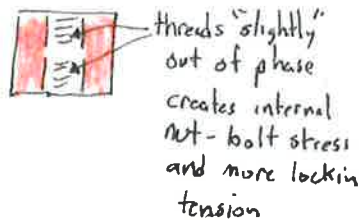


Self locking

Many ways to increase rotational static friction without bolt tension

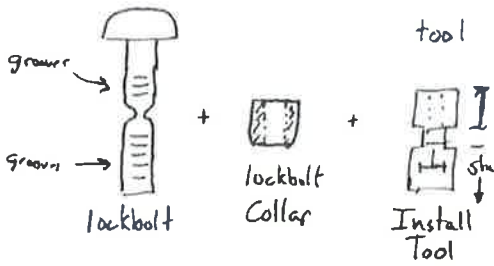


fiber, plastic insert

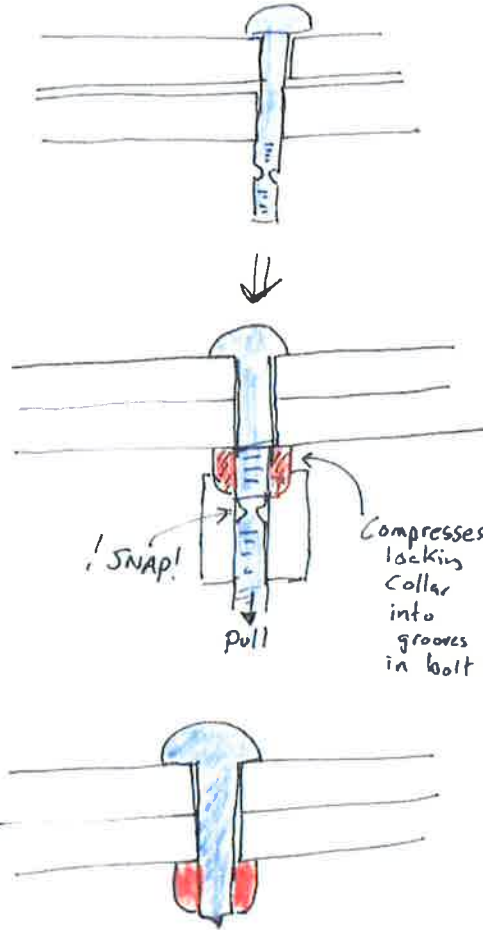


threads "slightly" out of phase creates internal nut-bolt stress and more locking tension

Lockbolt (bolt + rivet)



=



ALPP H T 8 8
 head fit pin diameter grip
 type "interlock" type 1/32" length
 pan head Steel 1/16"

BL 8 4
 blind lockbolt
 x "pop" rivet cousin

LC C 8
 lockbolt material diameter
 collar 1/32"

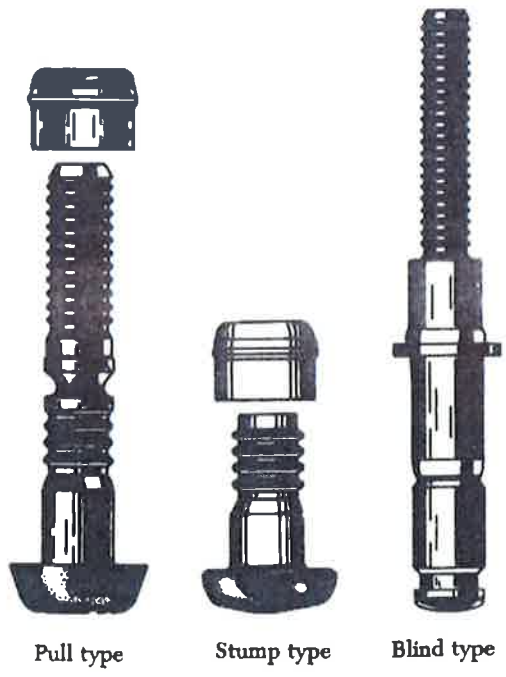


FIGURE 6-2. Lockbolt types.

Dzus "Zeus", Camloc, Airloc

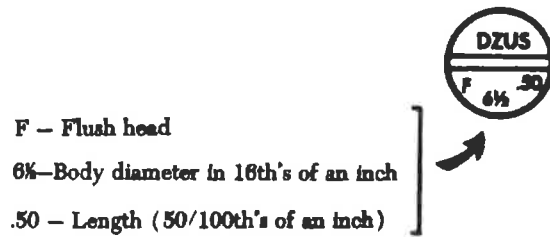
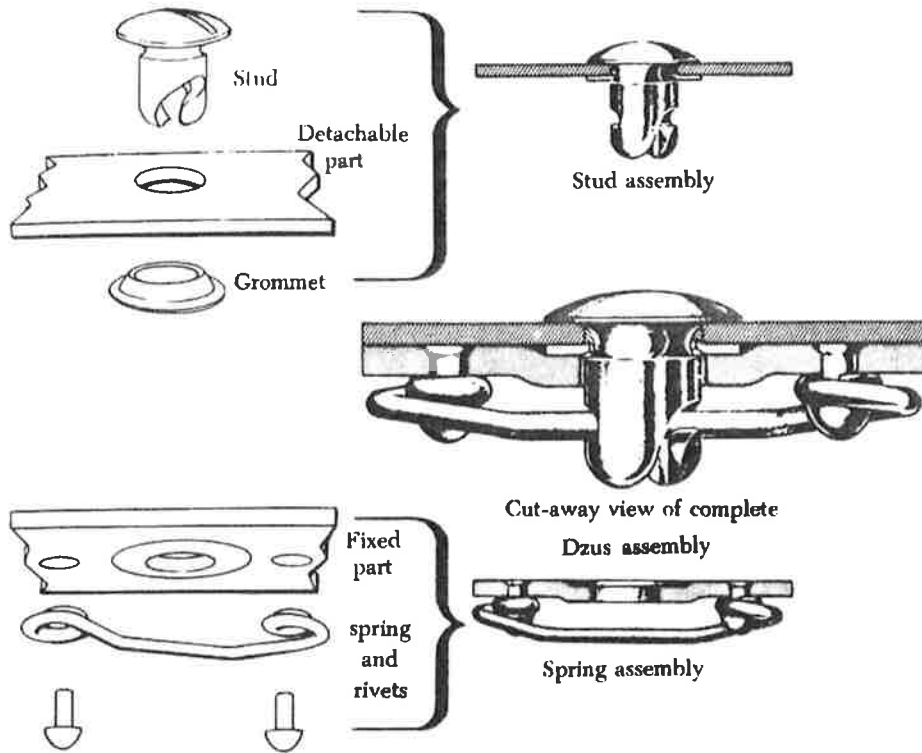
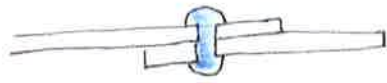
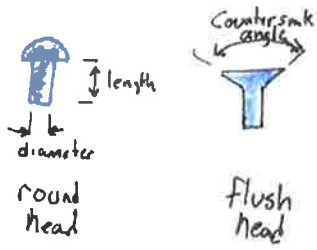


FIGURE 6-18. Dzus identification.

Rivets

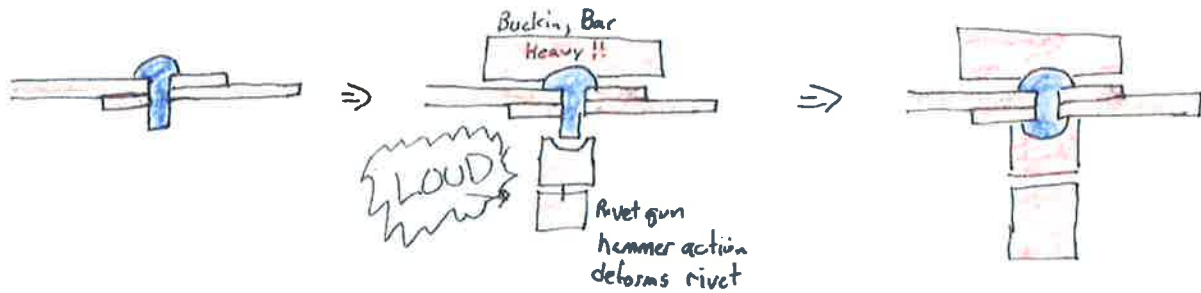


Light and strong joiner of sheet metal
(fast and cheap too!)



tiny.cc/AEM617Rivets
7:40

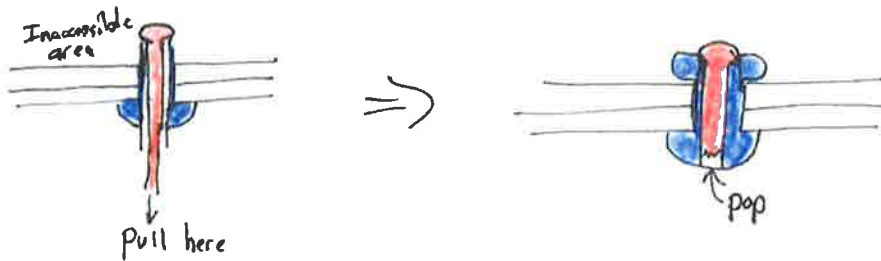
Install:



Heat Treating:

Some rivets require a special heat treating process before and during install.

"Pop" Rivets (call them blind rivets...)



Heavy, expensive, less strength, very convenient!

Material	Head Marking	AN Material Code	AN425 7/8 Counter- Sunk Head	AN496 1007 Counter- Sunk Head MS20428 *	AN427 100 Counter- Sunk Head MS20427 *	AN430 Round Head MS20470 *	AN435 Round Head MS20613 *	AN441 Flat Head	AN442 Flat Head MS20470 *	AN455 Brazier Head MS20470 *	AN456 Brazier Head MS20470 *	AN470 Universal Head MS20470 *	Heat Treat Before Using	Shear Strength P.S.I.	Bearing Strength P.S.I.
1100	Plain	A	X	X		X			X	X	X	X	No	10000	25000
2117T	Recessed Dx	AD	X	X		X			X	X	X	X	No	30000	100000
2017T	Raised Dx	D	X	X		X			X	X	X	X	Yes	34000	113000
2017T-HD	Raised Dx	D	X	X		X			X	X	X	X	No	39000	126000
2024T	Raised Double Dash	DD	X	X		X			X	X	X	X	Yes	41000	136000
5056T	Raised Cross	B	X	X		X			X	X	X	X	No	27000	90000
7075-T73	Three Raised Dashes		X	X		X			X	X	X	X	No		
Carbon Steel	Recessed Triangle				X		X						No	35000	90000
Corrosion Resistant Steel	Recessed Dash	F			X		X						No	65000	90000
Copper	Plain	C			X		X		X				No	23000	
Monel	Plain	M			X			X					No	49000	
Monel (Nickel-Copper Alloy)	Recessed Double Dash	C					X						No	49000	
Brass	Plain						X						No		
Titanium	Recessed Large and Small Dx			MS 20426			X						No	95000	

* New specifications are for Design purposes

FIGURE 6-33. Rivet identification chart.