

100 total points

Read, think, plan, and then write.

University of Alabama Academic Honor Pledge:

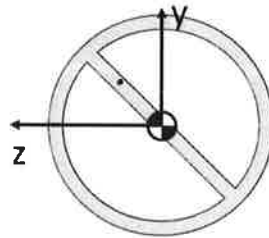
*I promise or affirm that I will not at any time be involved with cheating, plagiarism, fabrication, or misrepresentation while enrolled as a student at The University of Alabama. I have read the Academic Honor Code, which explains disciplinary procedures that will result from the aforementioned. I understand that violation of this code will result in penalties as severe as indefinite suspension from the University.*

Signature: \_\_\_\_\_ Date: \_\_\_\_\_

1. [10 pts] What is the sign (+, -, or zero) of  $I_{yz}$  for the following part around the given centroid?

$$I_{yz} = \int yz \, dA$$

+ + +  
- - +  
+



|              |              |              |                      |                   |
|--------------|--------------|--------------|----------------------|-------------------|
| $I_{yz} > 0$ | $I_{yz} < 0$ | $I_{yz} = 0$ | $I_{yz} = \pi r^3 t$ | None of the above |
|--------------|--------------|--------------|----------------------|-------------------|

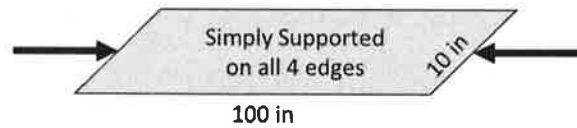
2. [10 pts] An applied load in the z-direction can result in beam tip **rotation** about x, y and z.

|      |       |                              |                             |                             |
|------|-------|------------------------------|-----------------------------|-----------------------------|
| True | False | Only for a uniform load in z | True only if $I_{yz} = 0$ ✗ | True only if $I_{yz} > 0$ ✗ |
|------|-------|------------------------------|-----------------------------|-----------------------------|

3. [10 pts] Why do spacecraft and aircraft use stringers and longerons? Discuss from a stress and buckling perspective.

- We want light structures. Thus, thin walled structures with*
- stringers for additional  $m_oI$*
  - stringers for increasing the buckling load.*

4. [35 pts] Will the flat panel shown below **break** or **buckle** first? The Al beam has a length of 100 inches, a width of 10 inches, and a thickness of 0.125 inches. The ultimate stress (i.e. break) is 40 ksi.



Buckling

$$k = 3.62 \quad \rightarrow \quad \frac{a}{b} = 10$$

$$P = t b k E \left(\frac{t}{b}\right)^2$$

$$\sigma = k E \left(\frac{t}{b}\right)^2 = 3.62 \cdot 10 \times 10^6 \text{ psi} \cdot \left(\frac{0.125}{10}\right)^2$$

$$= 5.6 \text{ ksi} < 40$$

Buckles

~~wrong type of buckling~~  
-15

wrong type of breaking  
-15

wrong BCs of buckling  
-10

right answer  
+5

5. [35 pts] Determine the axial stress on the top of the upper Al skin for the cross section below. The part has an applied load of  $Mz = 50 \cdot 10^3$  lbf-in and is heated +100 F. The foam's modulus of elasticity is 4 ksi. The foam's coefficient of thermal expansion is  $1.0 \cdot 10^{-5}$  1/F.

Fast way: Foam is inconsequential  $4 \text{ ksi} \ll 10 \times 10^6 \text{ psi}$

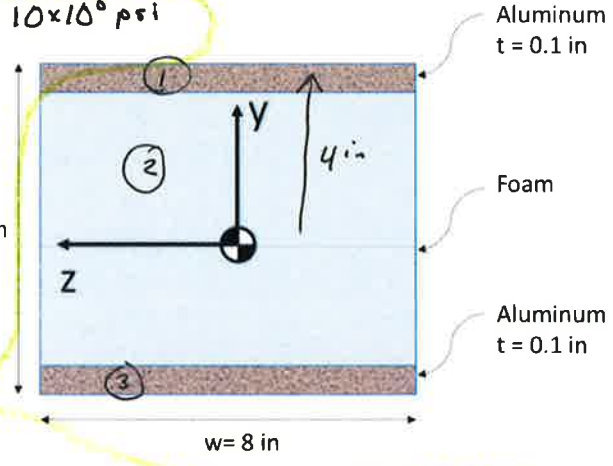
$$I_{zz} = \bar{y}^2 A \cdot 2 \text{ (only considering Al parts!)}$$

$$= 4^2 \cdot 0.1 \cdot 8 \cdot 2 = 25.6 \text{ in}^4$$

$$\sigma = \frac{-My}{I} = \frac{50000 \text{ lbf-in} \cdot 4 \text{ in}}{25.6 \text{ in}^4} =$$

$$\sigma = -7.8 \text{ ksi}$$

Thermal is dominated by Al, so ignore!



DRAWING NOT TO SCALE

Slower way (lumped)

| part         | E/E    | A    | $\bar{y}$ | $\bar{z}$ | $\frac{E}{E} (I_{zz} + \bar{y}^2 A)$ |
|--------------|--------|------|-----------|-----------|--------------------------------------|
| 1            | 1      | 0.8  | 4         |           | 12.8                                 |
| 2            | 0.0004 | 60.8 | 0         |           | 0                                    |
| 3            | 1      | 0.8  | -4        |           | 12.8                                 |
| $A^* = 1.62$ |        |      |           |           | $I_{zz} = 25.6$                      |

$$P_T = E \alpha \Delta T A$$

$$= 10 \times 10^6 \cdot 13 \times 10^{-6} \cdot 100 \cdot 8 \cdot 0.1 \cdot 2$$

$$+ 4000 \cdot 1 \times 10^{-5} \cdot 100 \cdot 8 \cdot 7.8$$

$$21049.6 \text{ lbf}$$

$$\sigma_{xx} = \frac{E}{E} \left[ \frac{21049.6}{1.62} - \frac{50000 \cdot 4 \text{ in}}{25.6} - 10 \times 10^6 \cdot 13 \times 10^{-6} \cdot 100 \right]$$

$$12993 - 7.8 - 13000$$

$$\sigma_{xx} = -7.8 \text{ ksi}$$

$$I_{zz}^* \text{ Alternative} = 62725$$

$$A^* = 4062.4$$

Very slow method

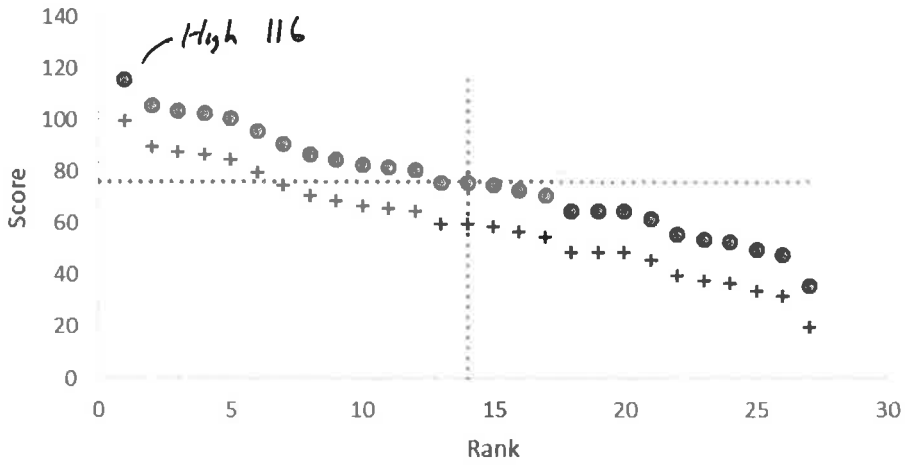
| part | E/E    | A    | $\bar{y}$ | $I_{zz}$ | $\frac{E}{E} (I_{zz} + \bar{y}^2 A)$ |
|------|--------|------|-----------|----------|--------------------------------------|
| 1    | 1      | 0.8  | 3.95      | 0.000667 | 12.48                                |
| 2    | 0.0004 | 60.8 | 0         | 316.4    | 0.126                                |
| 3    | 1      | 0.8  | -3.95     | 0.000667 | 12.48                                |
|      |        |      |           |          | $25.09$                              |

$$\sigma_{xx} = 12993 - \frac{50000 \cdot 4}{25.09} - 13000$$

$$-7.98 \text{ ksi}$$

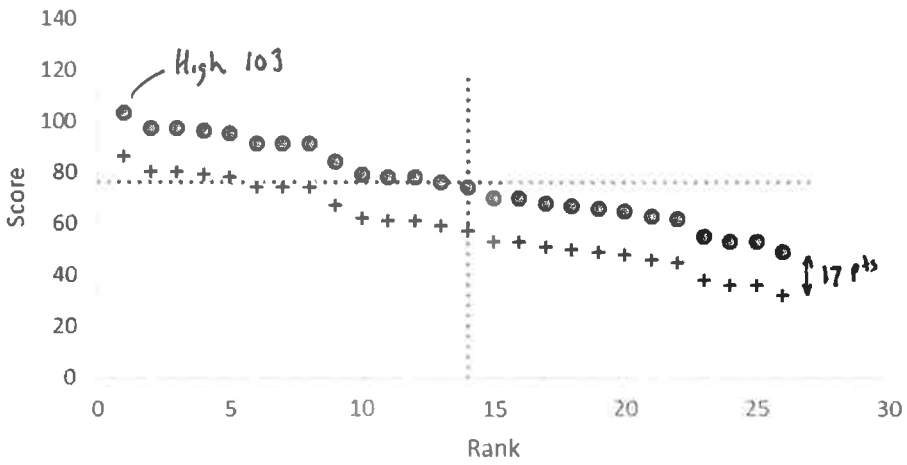
|                         |     |
|-------------------------|-----|
| $\frac{E}{E}$           | +2  |
| $\frac{I_{yy}}{I_{zz}}$ | +1  |
| $\bar{y}$               | +1  |
| $E \alpha \Delta T$     | +3  |
| $A^*$                   | +5  |
| $P_T$                   | +5  |
| $I_{zz}^*$              | +5  |
| $\sigma_{xx}$           | +10 |

### AEM 341 Exam 1



Avg 60  
 Shifted +16  
 Avg shifted = 76.3

### AEM 341 Exam 2



Avg 59  
 shifted +17  
 Avg shifted = 76

Approx:  
 Visual distribution

A B C