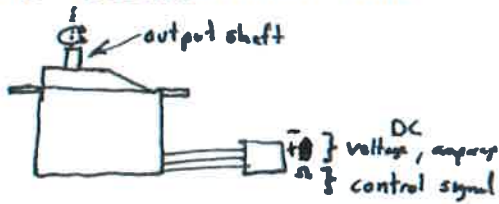


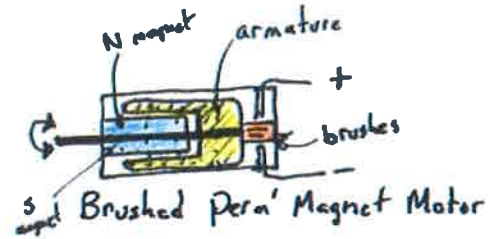
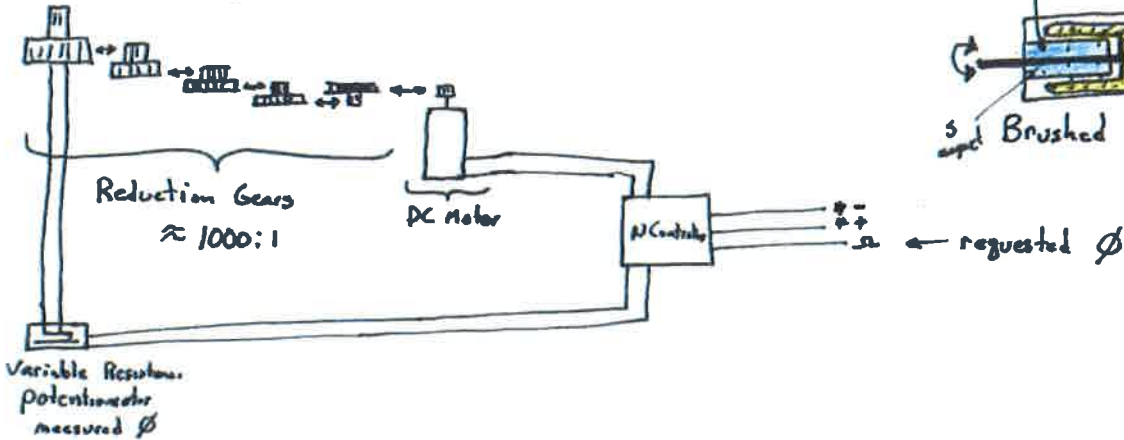
Servos
RC FCS
µC

Notes posted at: tiny.cc/ServoLecture

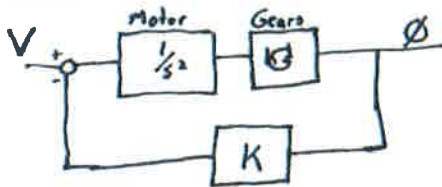
Analysis of the electromechanical servo.



Schematic



System Model



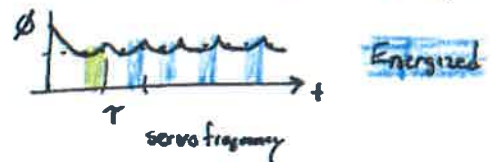
Simplifications:

- Gears have inertia
- Gears have play + friction
- Potentiometer is noisy (sometimes terrible on small servos)

Limitations

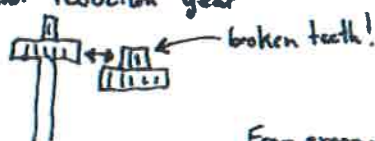
- Holding torque: An actual application requires holding a position ϕ with an applied torque.

The brushed motor can only provide torque when ~~the~~ current is applied. And the motor torque is more than necessary to maintain position ϕ . (i.e. overshoot). This is the source of servo buzz.



- Gear strength

The reduction drive requires high torque with a limited (very limited!) number of engaged teeth. High quality servos use metal gears or reinforced composites for the final reduction gear

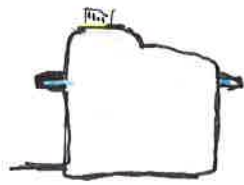


A true weak link

From experience, don't buy nylon geared servos.

Common Servo Sizes

see: servocity.com

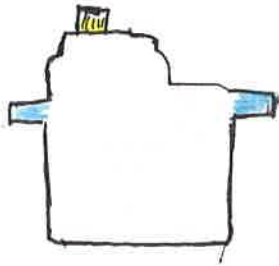


4.7g
≈ 5.0g

"Nano"

HS 35 HD

11 oz-in

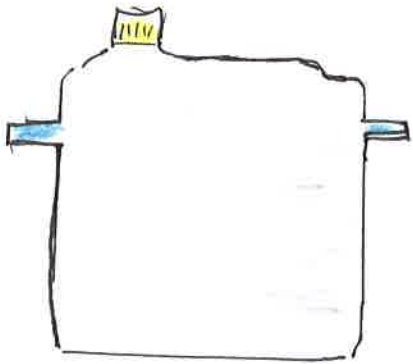


≈ 10g

Mini

HS 65 MG

25 oz-in @ 4.8v
31 oz-in @ 6.0v

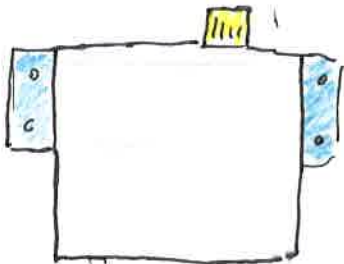


≈ 40g

Standard

Futaba S3003

44 oz-in @ 4.8v
57 oz-in @ 6.0v



Flat for thin aileron servos mounted in the wing.

Servo Signals

Pulse Width Modulation

"The width of a pulse specifies the servo angle"

The pulses have a 50 Hz PRF.

pulse repetition frequency



Since the period is the inverse of frequency,

$$T = \frac{1}{PRF} = \frac{1}{50 \text{ Hz}} = 0.02 \text{ s} = 20 \text{ ms}$$

A new pulse starts every 20ms

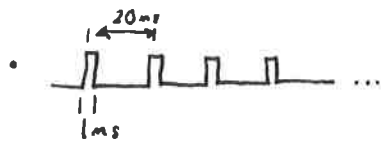
The servo angle is specified by the pulse width

where a 1ms pulse is the minimum angle and

a 2ms pulse is the maximum angle



Ex: We have a 90° travel servo. Show the angle specified by each pulse train.



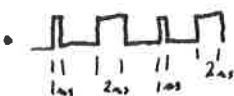
$$1 \text{ ms} \Rightarrow 0^\circ$$



$$1.5 \text{ ms} \Rightarrow 45^\circ$$



$$2.0 \text{ ms} \Rightarrow 90^\circ$$



Servo response is not 50Hz



Non compliant, response depends on servo controller ?

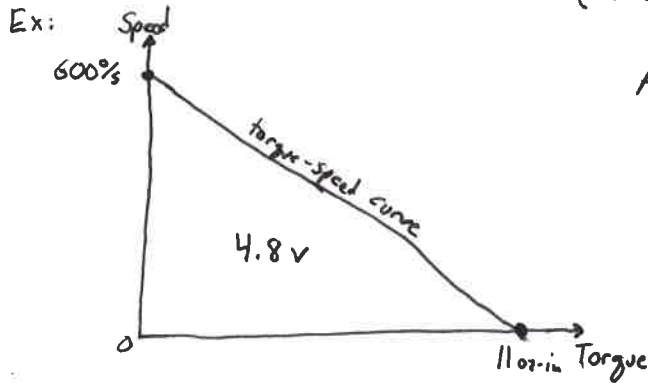
Servo Torque and Speed

Servos are effectively rated by torque and speed.

- Output "stall" torque (e.g. 11 oz-in)
- Maximum rotating speed (e.g. 60°/0.1s)

Because the servo is a motor, you won't get the maximum torque and the maximum speed at the same time.

Rather, there is a torque-speed curve (at a particular voltage)



Assume a straight line model:

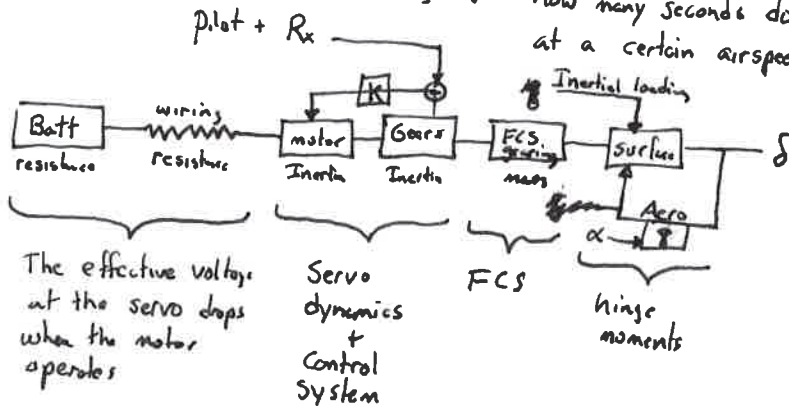
$$S(T) = 600\% - \frac{600\%}{11 \text{ oz-in}} \cdot T$$

If you need 400%, the maximum applied torque is:

$$400\% = 600\% - \frac{600\%}{11 \text{ oz-in}} \cdot T \Rightarrow T =$$

$$3.6 \text{ oz-in}$$

The question that we should be asking is "How many seconds does it take to get to full deflection at a certain airspeed?"



Hard Question

Q: Is this a reversible FCS? 368 topic
A: No.

Lower bound: Motor only with simple aero loads

Upper bound:

still an ODE $\dot{\theta} = f(T) = \dot{\theta}_{max} - \frac{\dot{\theta}_{max}}{T_{max}} T$
Determine the maximum torque at δ , assume torque constant over travel

Engineering Managers bound.

If you are operating in a region where the servo travel time is requiring you to determine a high quality hinge moment and motor model, you are operating too close to the edge.

Get a bigger servo! and Check your hinge moments (with analysis!)

HS-35HD ULTRA NANO SERVO	
SPECIFICATIONS	
AT 4.8Volt Only	
Operating Speed	0.1sec/60°
Output Torque	0.8kg-cm (11.11oz-in)
Weight	4.5g(0.158oz)
Size	18.8 X 7.6 X 15.5mm(0.73 X 0.30 X 0.61in)
FEATURES	
- Outstanding Lifespan (by dedicated potentiometer and DMK gears)	
- DMK (Dual Metal Karbonite) Gears (3rd and 4th) + 3 Karbonite Combination	
- Multi Directional Wiring System / - Ultra Light Weight	
PARTS & ACCESSORIES	
#56408 : HS-35HD / HS-5035HD Heavy Duty Gear Set	
#56409 : HS-35HD / HS-5035HD Case Set	
#55708 : HS-50/55/45/45/45/50/55MG/35HD/5035HD Horn Set	
#54657 : HS-35HD / 5035HD Light Weight Male Servo Connector(250mm)	

"Damasch" paper
"Root" paper
 $C_h \approx \frac{d C_h \delta}{d \delta}$
 $\delta = f(\theta)$

$H = g S c C_h$
Easy to solve once you select the models.

$T = f(G, \delta)$

Servo Gears

How I learned ^{and} to stop worrying
or Use quality equipment

There are essentially 3 types of servo gears:

Nylon / plastic :

Quiet, light, cheap, teeth fail easily

low torque applications

Reinforced plastic ("Karbonite" from Hitec)

Quiet, medium price, tough

medium torque

Metal :

Noisier, \$\$, wears out

High torque

More Torque?

Landing gear ... use a jackscrew



Slow!

Brands to consider

Futaba

Hitec

Spectrum

Fake Servos exist.

Amazon!

You can pay the same for a crummy servo.
(e.g. SPA story)

Avoid no-name
e-max
anything with poor typography

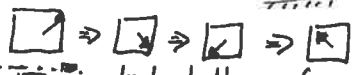
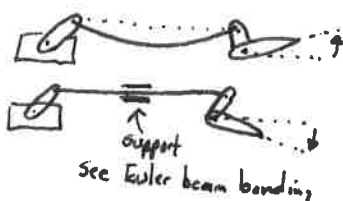
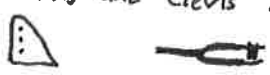
Deadband / hysteresis



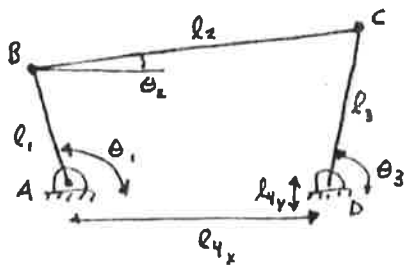
Unusable
for aircraft

poor frequency response

Flight Control System (FCS) mistakes leading to flight path deviations

- Reversed Controls \Rightarrow preflight, "box" controls 
- Weak or dead battery \Rightarrow preflight check + timer ^{or timer} look at the surfaces!
size the battery for current draw
- Broken antenna or poor reception \Rightarrow range check, dipole orientation
- Interfering controls (e.g. rudder w/ up elevator) \Rightarrow "box" controls + preflight
- Stripped Gears after hard landing \Rightarrow inspect and test each component after hard landings
- High g hinge moments (esp. elevator or tilt) \Rightarrow proper sizing of servo
(in a dive, less up elevator allows g to increase even more)
- Control surface flutter (saturates servo... loss control) \Rightarrow stress/deflection of surfaces especially torsion.
- Frequency Interference (esp. other 72 MHz radios) \Rightarrow test on ground + preflight
- Wrong mode (e.g. mismatch between brain + aircraft) \Rightarrow human factors + checklist
- Foreign Object interference with Tx, Rx, or FCS \Rightarrow clean FCS area
(e.g. neck strap jamming rudder) ^{true case}
(e.g. linkage at g loading w/ vibration w/ other control)
clean pilot
remove extra junk + test
- ^{fake!} poor quality servos or wrong type of servos \Rightarrow buy quality and do the analysis
- Bending linkages  \Rightarrow proper support, pull-pull system
horn mounting with back plate
- Wrong or poorly tuned feedback control gains \Rightarrow gnd demo, check for sign error, analysis
- loose or broken horns and clevis or missing screws \Rightarrow preflight + airworthiness check (3rd party?)

- PIO (pilot induced oscillations), control power issues \Rightarrow Analysis w/ #s, $\eta = \frac{g}{g_{00}}$
- Aft CG, not enough dihedral, S+C issues \Rightarrow Analysis w/ #s
(e.g. Falcon, Spata) see within seconds!
Design reviews, listen to others

General 4 bar



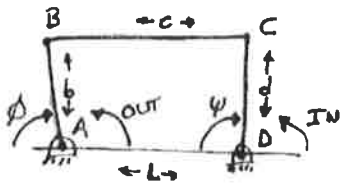
Assume system is connected

$$X \text{ direction: } l_1 \cos \theta_1 + l_2 \cos \theta_2 + l_3 \cos \theta_3 - l_{4x} = 0$$

$$Y \text{ direction: } l_1 \sin \theta_1 + l_2 \sin \theta_2 + l_3 \sin \theta_3 - l_{4y} = 0$$

These are constraints but they don't give us information on $\theta_1 = f(\theta_3)$

Freudenstein Equation (notice the angle convention!)



Consider the term AB as the vector from A to B

$$AB = \uparrow \quad CD = \downarrow \quad DC = \uparrow$$

From continuity, $AB + BC = AD + DC$

Solve for $BC = AD + DC - AB = -(DA + CD + AB)$

Freudenstein's insight was to take the dot product of BC with BC.

$$BC \cdot BC = -(DA + CD + AB) \cdot -(DA + CD + AB)$$

Find terms for location of B, C (assuming mechanism is aligned with x axis at A and D)

$$B = [-b \cos \phi, b \sin \phi] \quad C = [-d \cos \psi, d \sin \psi]$$

Simplify $BC \cdot BC = c^2$!! Such that

$$c^2 = [1 + b \cos \phi - d \cos \psi, -b \sin \phi + d \sin \psi] \cdot [d, b]$$

After some simplification

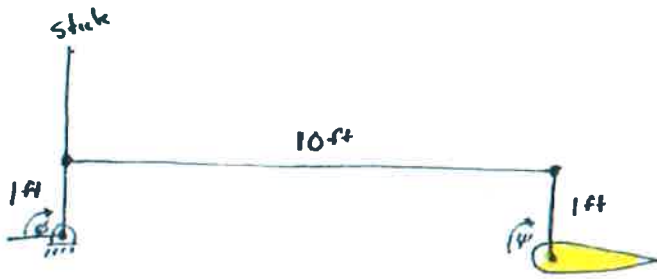
$$R_1 \cos \phi = R_2 \cos \psi + R_3 = \cos(\phi - \psi)$$

$$R_1 = \frac{c^2}{d} \quad R_2 = \frac{c^2}{b} \quad R_3 = \frac{L^2 + b^2 - c^2 + d^2}{2bd}$$

4 unknowns

4 equations \rightarrow You need to specify 4 positions.

Example



$L = 10 \text{ ft}$
 $c = 10 \text{ ft}$
 $b = 1 \text{ ft}$
 $d = 1 \text{ ft}$

$$R_1 = \frac{L}{d} = 10$$

$$R_2 = \frac{L}{b} = 10$$

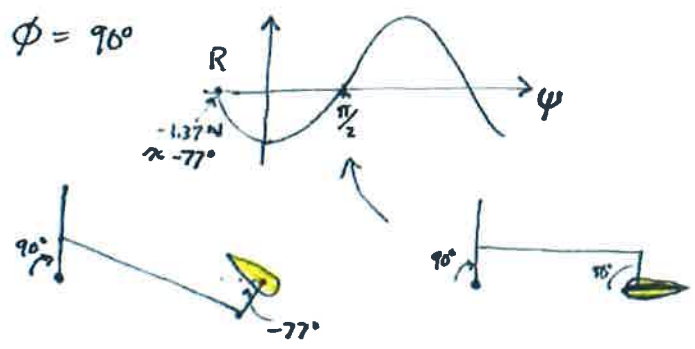
$$R_3 = \frac{100 + 1 - 100 + 1}{2} = 1$$

Freudenstein

$$10 \cos \phi - 10 \cos \psi + 1 = \cos(\phi - \psi)$$

$$\text{Residual} \equiv 10 \cos \phi - 10 \cos \psi + 1 - \cos(\phi - \psi) \rightarrow 0$$

when $\phi = 90^\circ$



What about the gearing ratio? $\frac{d\psi}{d\phi}$

Take $\frac{d}{d\phi}$ of Freudenstein while remembering that $\psi = f(\phi)$

$$\frac{d}{d\phi} (R_1 \cos \phi - R_2 \cos \psi + R_3 - \cos(\phi - \psi))$$

$$= -R_1 \sin \phi \frac{d\phi}{d\phi} + R_2 \sin \psi \frac{d\psi}{d\phi} + 0 + \sin(\phi - \psi) \underbrace{\frac{d}{d\phi}(\phi - \psi)}_{1 - \frac{d\psi}{d\phi}}$$

Solve for $\frac{d\psi}{d\phi}$

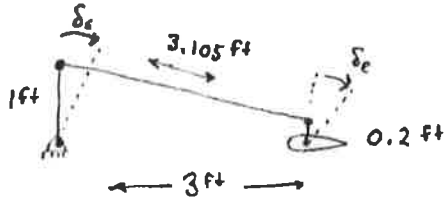
$$\frac{d\psi}{d\phi} = \frac{-\sin(\phi - \psi) + R_1 \sin \phi}{R_2 \sin \psi - \sin(\phi - \psi)}$$

For the above,

$$\frac{d\psi}{d\phi} (\theta = 90, \psi = 90) = 1.0 \quad \checkmark$$

$$\frac{d\psi}{d\phi} (\theta = 90, \psi = \pi) \approx -0.98$$

Example (shorter pushrod)



$$L = 3$$

$$b = 1$$

$$c = 3.105$$

$$d = 0.2$$

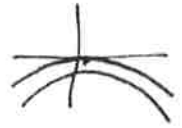
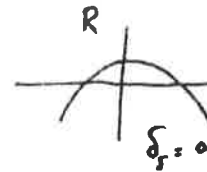
$$R_1 = 15 \quad R_2 = 3 \quad R_3 = 0.9974$$

At $\delta_s = 0$ We expect a gearing ratio of 5, since $\frac{b}{d} = 5$

What are the limits of stick travel?

$$\delta_s \approx 15^\circ \Rightarrow \delta_e = 108^\circ$$

$$\delta_s \approx -9^\circ \Rightarrow \delta_e = -70^\circ$$



What is the gearing ratio at $\delta_s = 15^\circ$? $G \approx 216$

" at $\delta_s = -9^\circ$ $G \approx 380$

← This indicates that 1 lb-ft at the surface requires 200-300 ft-lb at the stick ... not happening ...

